

Mathematical modelling on particle diffusion in fluidised beds and dense turbulent two-phase flows

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Abstract

Volume-fraction weighted and Reynolds averaged momentum transport equations are solved in an Euler/Euler approach to numerically simulate the turbulent, dispersed two-phase flow in a two-dimensional channel and a three-dimensional conic diffuser flow. Particular attention is given to the modelling of turbulent diffusion and particle wall interaction, assuming local equilibrium, but introducing individual terms for particle/fluid drag interaction, particle collisions and trajectory crossings. These influences have been quantified in terms of partial viscosities, a restitution power and a turbulence structure parameter. Boussinesq approximations have been used for each phase and the formulation of their interaction was provided in the framework of the eddy-viscosity modelling concept.

Keywords: two-phase flow, particle diffusion, particle collision, fluidised bed.

1 Introduction

The momentum transport equation includes a turbulent diffusion term, which characterises motions that are not resolved by the convective term. This turbulent diffusion depends on the turbulent kinetic energy and the turbulence characterising Eulerian time scale. Based on "Csanadys Approximation" [3], the time scale of the dispersed phase is coupled with the turbulent time scale of the continuous phase. This time scale quantifies the diffusion intensity [5] and is influenced by the drag interaction of the particles with the viscous gas phase and inter-particle collisions [4, 9]. The fluctuation of a filtered variable ϕ is written:

$$\{\phi\}^k = \phi - \langle \phi \rangle^k . \quad (1)$$



The index k stands for C (continuous phase) or D (dispersed phase). The following negation notation is used here, i.e. $\bar{C} = D$, $\bar{D} = C$. The drag force relaxation time scale τ_p^k depends on the particle Reynolds number, based on the velocity difference between the continuous and the dispersed phase [12], see e.g. Crowe et al. (1998) [2]. The equation terms are described with the help of an averaging operator $\langle \cdot \rangle^k$ [11]:

$$\langle u_i^k \rangle^l = \frac{\alpha^l u_i^k}{\bar{\alpha}^l}; \quad u_i^k = \langle u_i^k \rangle^l + \{u_i^k\}^l. \quad (2)$$

Summarising the different diffusion generating components to a combined particle-turbulence shear stress model the diffusion of dispersed particles is predicted very well, especially inside turbulent shear layers.

2 Transport equations

Modelling the diffusion character of particle flows transported in a viscous carrier phase the diffusion is not dominated by the eddy dissipation as it is in turbulent shear flows in a continuous gas phase. This dispersed phase diffusion is characterised by unsteady drag influences and crossing trajectory characteristics defined by the correlation of velocities of the continuous phase and the dispersed phase itself. The Reynolds averaged, volume-fraction weighted momentum transport equations for the continuous and the dispersed phase read:

$$\begin{aligned} & \frac{\partial}{\partial t}(\rho^k \bar{\alpha}^k \langle u_i^k \rangle^k) + \frac{\partial}{\partial x_j}(\rho^k \bar{\alpha}^k \langle u_i^k \rangle^k \langle u_j^k \rangle^k) \\ &= -\bar{\alpha} \frac{\partial p^C}{\partial x_i} + \bar{\alpha} \frac{\partial \sigma_{ij}^k}{\partial x_j} + \rho^k \frac{\bar{\alpha}^k}{\tau_p^k} (\langle u_i^{\bar{k}} \rangle^{\bar{k}} - \langle u_i^k \rangle^k) \\ &+ \rho^k \bar{\alpha}^k g_i - \frac{\partial}{\partial x_j}(\rho^k \bar{\alpha}^k \langle \{u_i^k\}^k \{u_j^k\}^k \rangle^k) \end{aligned} \quad (3)$$

Collision and crossing trajectory terms were approximated by Grad [4] and Csanady [3]. The pressure gradient of the continuous phase is equivalent to the lift force. And the sum would be zero in a hydrostatic case. σ_{ij}^k defines the viscous stresses inside the phase k . Because of the solid character of the dispersed phase σ_{ij}^D is zero.

The underlying turbulence model for both phases is based on the *Boussinesq analogy*, employing eddy viscosity as the model quantity, whose formulation was provided in the framework of the standard $k - \epsilon$ modelling concept. k^C defines the turbulent kinetic energy of the continuous phase. The corresponding variable of the dispersed phase k^D describes the particle velocity variance at a given position.



$$\begin{aligned}
& \frac{\partial}{\partial t}(\rho^k \bar{\alpha}^k k^k) + \frac{\partial}{\partial x_j}(\rho^k \bar{\alpha}^k k^k < u_j^k >^k) \\
&= \frac{\partial}{\partial x_j} \left(\bar{\alpha}^k \left(\mu^k + \rho^k \frac{v_t^k}{\sigma_k^k} \right) \frac{\partial k^k}{\partial x_j} \right) - \rho^k \bar{\alpha}^k < \{u_i^k\}^k \{u_j^k\}^k >^k \frac{\partial < u_i^k >^k}{\partial x_j} \\
&\text{with } \sigma_k^C = 1, 0 \text{ and } \sigma_k^D = 2.5
\end{aligned} \tag{4}$$

The transport equations for the turbulent kinetic energy k^k and its dissipation rate ϵ_t^k differ from those for a single-phase flow by several additional production terms and the modified dissipation. This total turbulent kinetic energy loss ϵ_α^k in the present model arises from relative drag and particle collision processes.

$$\begin{aligned}
& \frac{\partial}{\partial t}(\rho^k \bar{\alpha}^k \epsilon_t^k) + \frac{\partial}{\partial x_j}(\rho^k \bar{\alpha}^k \epsilon_t^k < u_j^k >^k) \\
&= \frac{\partial}{\partial x_j} \left(\bar{\alpha}^k \left(\mu^k + \rho^k \frac{v_t^k}{\sigma_\epsilon^k} \right) \frac{\partial \epsilon_t^k}{\partial x_j} \right) - C_1 \frac{\epsilon_t^k}{k^k} \rho^k \bar{\alpha}^k < \{u_i^k\}^k \{u_j^k\}^k >^k \frac{\partial < u_i^k >^k}{\partial x_j} \\
&\quad - C_2 \rho^k \bar{\alpha}^k \frac{\epsilon_t^k \epsilon_\alpha^k}{k^k} + C_3 \frac{\epsilon_t^k}{k^k} \rho^k \frac{\bar{\alpha}^k}{\tau_p^k} (q^k - 2k^k) \\
&\text{with } \sigma_\epsilon^C = 1.3, C_1 = 1.44, C_2 = 1.92, C_3 = 1.2
\end{aligned} \tag{5}$$

Because of the different phase velocities inside the drag relation term a general formulation is needed for the complimentary index of k . The velocity covariance $q^k = < \{u_i^k\}^k \{u_i^{\bar{k}}\}^{\bar{k}} >^k$ of He and Simonin (1993) [5] represents the trace of the velocity vector correlation tensor of both phases:

$$\begin{aligned}
& \frac{\partial}{\partial t}(\rho^k \bar{\alpha}^k q^k) + \frac{\partial}{\partial x_j}(\rho^k \bar{\alpha}^k q^k < u_j^k >^k) \\
&= \frac{\partial}{\partial x_j} \left(\rho^k \bar{\alpha}^k v_\alpha^k \frac{\partial q^k}{\partial x_j} \right) + \rho^k \frac{\bar{\alpha}^k}{\tau_p^k} (2Z^k k^k + 2k^{\bar{k}} - (1 + Z^k) q^k) \\
&\quad - \rho^k \bar{\alpha}^k < \{u_i^k\}^k \{u_j^k\}^k >^k \frac{\partial < u_i^k >^k}{\partial x_j} - \rho^k \bar{\alpha}^k < \{u_i^{\bar{k}}\}^{\bar{k}} \{u_j^{\bar{k}}\}^{\bar{k}} >^k \frac{\partial < u_i^{\bar{k}} >^{\bar{k}}}{\partial x_j} \\
&\text{with } Z^k = \frac{\rho^k \bar{\alpha}^k}{\rho^{\bar{k}} \bar{\alpha}^{\bar{k}}},
\end{aligned} \tag{6}$$

which completes the present three-equation model for each of both phases. Velocity correlations, representing the turbulent momentum diffusion, are modelled by the following Boussinesq approximations:

$$\begin{aligned}
& - < \{u_i^k\}^k \{u_j^k\}^k >^k \\
&= -\frac{2}{3} k^k \delta_{ij} + v_t^k \left(\frac{\partial < u_i^k >^k}{\partial x_j} + \frac{\partial < u_j^k >^k}{\partial x_i} - \frac{2}{3} \frac{\partial < u_l^k >^k}{\partial x_l} \delta_{ij} \right)
\end{aligned} \tag{7}$$



$$\begin{aligned}
& - < \{u_i^k\}^k \{u_j^k\}^k >^k \\
& = -\frac{1}{3} q^k \delta_{ij} + v_\alpha^k \left(\frac{\partial < u_i^k >^k}{\partial x_j} + \frac{\partial < u_j^k >^k}{\partial x_i} - \frac{2}{3} \frac{\partial < u_l^k >^k}{\partial x_l} \delta_{ij} \right). \quad (8)
\end{aligned}$$

Modelling the viscous continuous phase the eddy viscosity is modelled based on eddy dissipation relating components. Because of the phase interaction influence on the turbulent diffusion of both phases an additional transport equation for the velocity covariance has to be defined. The diffusion coefficients of momentum and turbulent transport equations are given by the characteristic diffusion time scale τ_α^k :

$$v_t^k = \tau_\alpha^k \cdot \frac{2}{3} k^k \quad \text{and} \quad v_\alpha^k = \tau_\alpha^k \cdot \frac{2}{3} q^k. \quad (9)$$

To close the present formulation of particle and carrier gas phase motion this time scale has to be modelled. Based on this new kind of particle diffusion modelling also equilibrium-turbulence boundary conditions are modified by the influences of inter-particle collisions and phase-interactions of dispersed and continuous phase.

3 Diffusion modelling

Based on this model of momentum diffusion, which depends on the velocity gradients of the diffusing phase, the characteristic diffusion time scales have to be defined by the velocity correlation and its associated loss rate ϵ_α^k . Local equilibrium describes the equivalence of production and loss of turbulent kinetic energy. Assuming $\partial/\partial x_1 \approx 0$ yields an expression ($i = 1, j = 2$) for the non-diagonal element of the Reynolds stress tensor.

$$\begin{aligned}
\epsilon_\alpha^k &= - < \{u_1^k\}^k \{u_2^k\}^k >^k \frac{\partial < u_1^k >^k}{\partial x_2} \quad (10) \\
\Rightarrow \quad - < \{u_1^k\}^k \{u_2^k\}^k >^k &= \frac{(< \{u_1^k\}^k \{u_2^k\}^k >^k)^2}{\epsilon_\alpha^k} \frac{\partial < u_1^k >^k}{\partial x_2}
\end{aligned}$$

With the definition of the turbulence structure parameter C_α^k of the phase k

$$\begin{aligned}
C_\alpha^k &= - \frac{< \{u_1^k\}^k \{u_2^k\}^k >^k}{\frac{2}{3} k^k} \quad (11) \\
\Rightarrow \quad v_t^k \frac{\partial < u_1^k >^k}{\partial x_2} &= - < \{u_1^k\}^k \{u_2^k\}^k >^k = \frac{\left(\frac{2}{3} C_\alpha^k k^k\right)^2}{\epsilon_\alpha^k} \frac{\partial < u_1^k >^k}{\partial x_2}
\end{aligned}$$

the turbulent viscosity is calculated using the turbulent kinetic energy k^k , its loss rate and the turbulence structure parameter. Based on the diffusion definition (eq.9)



Table 1: Restitution power and turbulence loss components.

γ	model	π_γ^C	ϵ_γ^C	π_γ^D	ϵ_γ^D
β	Jones et al.	$\frac{2}{3}\beta\epsilon_\beta^C$	ϵ_β^C	0.	0.
d	Csanady	0.	0.	$\frac{2}{3}\frac{k^D}{\tau_\alpha^C}\sqrt{1+C_\beta^C(\xi_r^C)^2}$	0.
p	Wallis	$\frac{2}{3}\frac{2k^C-q^C}{\tau_p^C}$	$\frac{2k^C-q^C}{\tau_p^C}$	$\frac{2}{3}\frac{2k^D-q^D}{\tau_p^D}$	$\frac{2k^D-q^D}{\tau_p^D}$
c	Jenkins et al.	0.	0.	$\frac{\sigma_c^D}{\tau_p^D}\frac{2}{3}k^k$	$\frac{k^k}{\tau_c^D}(1-(e_c^D)^2)$

the turbulent time scale is also defined by these values:

$$v_t^k = -\frac{2}{3}\tau_\alpha^k k^k \Rightarrow \tau_\alpha^k = \frac{v_t^k}{\frac{2}{3}k^k} = \frac{2}{3}\left(C_\alpha^k\right)^2 \frac{k^k}{\epsilon_\alpha^k} \quad (12)$$

With the definition of the restitution power π_α^k , the turbulent viscosity is determined by the turbulent kinetic energy and the restitution power.

$$\pi_\alpha^k = \frac{2}{3}\frac{k^k}{\tau_\alpha^k} = \frac{\epsilon_\alpha^k}{(C_\alpha^k)^2} \Rightarrow -\langle \{u_1^k\}^k \{u_2^k\}^k \rangle^k = \frac{\left(\frac{2}{3}k^k\right)^2}{\pi_\alpha^k} \frac{\partial \langle u_1^k \rangle^k}{\partial x_2} \quad (13)$$

The power π_α^k describes the restitution of turbulent shear forces based on the dissipation and structure of turbulence and reduces the turbulent diffusion.

This restitution power consists of the partial powers defined by four different effects (see Table 1), which are described in the following subsections with the indices β , p , c and d . The total turbulence loss rate ϵ_α^k is given by the sum of individual loss rates (ϵ_γ^k , see Table 1):

$$\epsilon_\alpha^k = \sum_\gamma \epsilon_\gamma^k = \epsilon_\beta^k + \epsilon_d^k + \epsilon_p^k + \epsilon_c^k \quad (14)$$

The different diffusion rates and turbulence loss rates are induced by the viscous turbulent shear stress ($\pi_\beta^k, \epsilon_\beta^k$: Jones et al. [7]), crossing trajectory effects (π_d^k, ϵ_d^k : Csanady [3]), drag forces (π_p^k, ϵ_p^k : Schiller and Naumann [12]) and collision terms (π_c^k, ϵ_c^k : Jenkins and Richman [9]). Adding together these influences, the new restitution power term of the turbulent diffusion is modelled.

$$\pi_\alpha^k = \sum_\gamma \pi_\gamma^k = \pi_\beta^k + \pi_d^k + \pi_p^k + \pi_c^k \quad (15)$$



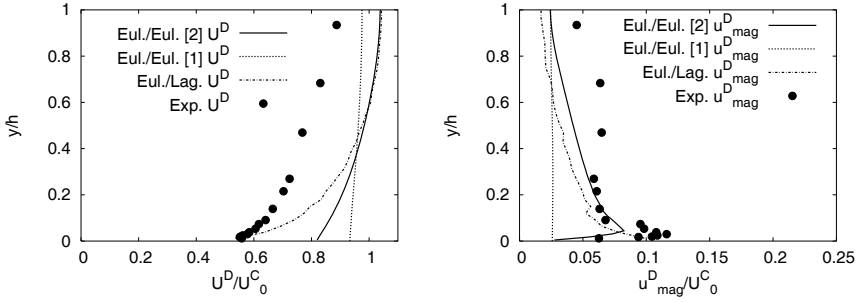


Figure 1: Normalised stream-wise particle mean velocity U^D and normalised standard deviation u_{mag}^D of the particle velocity magnitude in the fully-developed, particle-laden channel flow.

Simulating the general restitution power π_α^k and energy loss ϵ_α^k the resulting turbulence structure parameter C_α^k is computed in the following way:

$$\pi_\alpha^k = \frac{2}{3} \frac{k^k}{\tau_\alpha^k} = \frac{\epsilon_\alpha^k}{(C_\alpha^k)^2} \Rightarrow C_\alpha^k = \sqrt{\frac{\epsilon_\alpha^k}{\pi_\alpha^k}} \quad (16)$$

The turbulent time scale τ_α^k depends on the sum of all diffusion rates:

$$\tau_\alpha^k = \frac{\frac{2}{3} k^k}{\pi_\beta^k + \pi_d^k + \pi_p^k + \pi_c^k} \quad (17)$$

This way of calculation yields a deterministic method to compute the turbulent time scale, which is needed for the calculation of the general turbulent viscosity of both phases (eq.12).

4 Computational results and discussion

This model was validated using experimental data of 70 μm copper particles in a fully-developed channel flow ([10]; experimental results). The results using the present model (Eul./Eul. [2]) were also compared with the results obtained by an Euler/Lagrange scheme (Eul./Eul.) [6, 8] and a well-known Euler/Eulerian diffusion approach (Eul./Eul. [1]) [5, 11].

Gravity acts in the positive x-axis direction. The channel flow Reynolds number, based on channel height ($2h = 40 \text{ mm}$) and single phase channel centreline velocity ($U_0^C = 10.5 \text{ m/s}$) is $Re_{2h} = 27600$. The flow is regarded as fully developed after 125 channel heights and at this position it is assumed that the particle velocity and particle turbulence has reached an asymptotic state. The copper particles have a density of $\rho^D = 8800 \text{ kg/m}^3$ and a diameter of $D_p = 70 \mu\text{m}$. The inlet mass loading of particles is $Z_0^D = 10\%$ and the parameter of

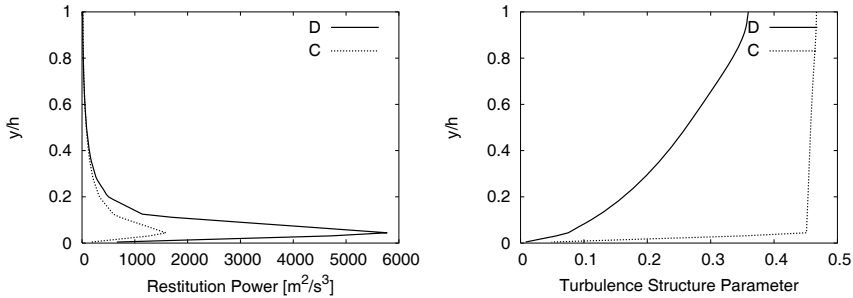


Figure 2: Turbulent kinetic energy loss rate ϵ_{α}^k and turbulence restitution power π_{α}^k of the dispersed phase (D) and the continuous phase (C).

elasticity is $e_c^D = 0.90$. The results obtained using the present method were also compared with the computational results obtained by an Euler/Lagrange scheme [8].

Comparing the present model (Eul./Eul. [2]) with a standard particle diffusion model (Eul./Eul. [1]), the velocity and the standard deviation of the velocity magnitude of the present model agree better with the Euler/Lagrangian results. Because of the assumed isotropy of the present model, the standard deviation of the stream-wise particle velocity does not agree as well with the predicted standard deviation of the particle velocity magnitude.

The characteristic model values describing diffusion: the restitution power π_{α}^k and the turbulence structure parameter C_{α}^k are shown in Fig. 2. The dissipation loss of the continuous phase and the restitution power of the dispersed phase dominate over the respective values of the other phase. As expected the turbulence structure parameter of the dispersed phase C_{α}^D decreases near the wall, because of the decreasing ratio $\sqrt{\epsilon_{\alpha}^D/\pi_{\alpha}^D}$ against the nearly constant turbulence structure parameter of the continuous phase C_{α}^C . Comparing the positions of these restitution maxima of the momentum diffusion with local minima of the turbulence structure parameter C_{α}^D (Fig. 2, right) near the wall the difference between the different kinds of diffusion, with and without viscous turbulence dissipation, in dispersed and continuous phase are shown.

Validating this new particle diffusion model a test case of Bohnet and Triesch [1] has been chosen. Simulating a fluidised bed with a particle loading $Z = 1$ inside a rising vertical diffuser flow. Glass particles are used approaching the dispersed phase. The half-cone angle of the conic diffuser is 6° and the outlet/inlet diameter ratio is $D_2/D_1 = 1.45$. The particle laden flow enters the diffuser part of the channel after a distance, which is long enough to develop a fully turbulent flow.

Analysing the velocity profiles (Fig. 3) local loading, gas and particle velocities are shown at entrance and exit of the diffuser. The gas phase inlet-boundary condition is a velocity block profile of $U_0 = 26.0$ m/s. The symmetry line is at $x = 0$. Inside the diffuser the gas velocity decreases and is overtaken by the particles. The particle velocity decreases during the relaxation process upside the

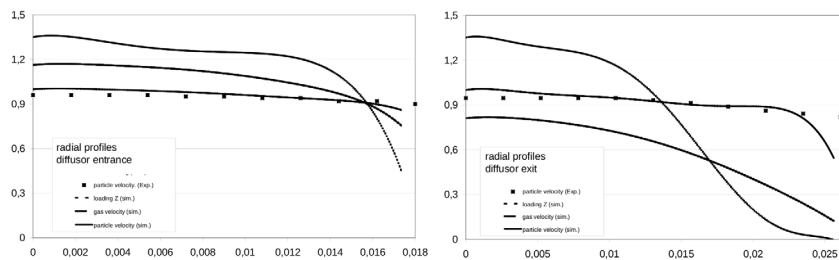


Figure 3: Normalised radial profiles of axial gas velocity, particle velocity and loading in comparison with experimental results.

diffuser. The particle loading has a similar maximum at the exit of the diffuser. Also this maximum decrease during the homogenising diffusion process upside the diffuser. The numerical data are compared with experimental data of Bohnet and Triesch [1].

5 Conclusions

Using this kind of diffusion blending, simulations are able to give better results for turbulent wall layers of the dispersed phase, including their turbulence production. The prediction of turbulent particle diffusion is limited by the quality of modelling of the momentum diffusion and the turbulence production of the dispersed phase.

Compared to classical equilibrium models, which solve an additional differential equation for the energy loss, coming up to the dissipation rate in viscous systems, the energy loss of the dispersed phase is given by algebraic equations. The turbulence structure parameter remains nearly constant in the dissipative systems examined here. So the ratio of restitution and dissipation power of the involved sub-models keeps also nearly constant.

For the not viscous, dispersed phase the turbulence structure parameter decreases corresponding to the high restitution and the locally low momentum diffusion inside the wall layer. This characteristic behaviour is based on the additional restitution power without the corresponding loss rate induced by crossing trajectories effects. Basically the added modelling of the turbulence structure parameter influences the production of particle velocity variance near the wall in that kind that the gradients of the stream-wise particle velocity agree with the measurement data.

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